

1.[2] 가 ( ) ,

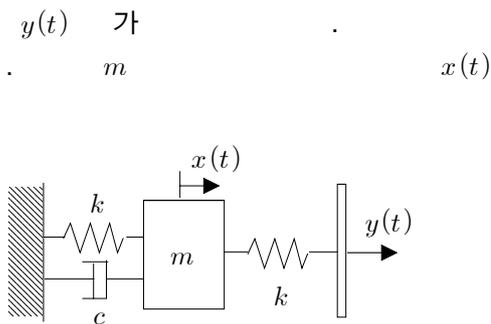
2.[6] O , X ( ) . ( )

(a) (guitar) (La) 440 Hz

(b) 가 ( )

(c) 가 ( )

3.[6] (damper) 2 1 가 m

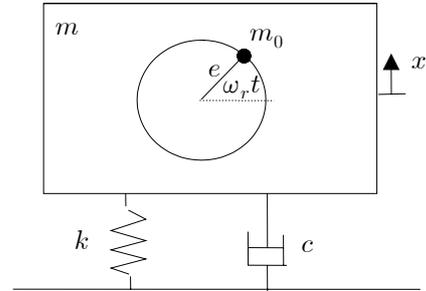


(a) m 가 (free-body diagram)

(b) 가 가  $y(t) = Y \sin(20\pi t)$  mm , 6.0 mm

$c = 3400$  N·s/m

4.[6] Consider a typical unbalanced machine problem as shown below with a machine mass of 120 kg, a mount stiffness of 500 kN/m, and a damping value of 4000 kg/s. The amplitude of the out-of-balance force is measured to be 350 N at a running speed of 3600 rev/min. The machine vibrates in the vertical direction.



(a) Determine the frequency ratio and damping ratio.

(b) Determine the amplitude of motion due to the out of balance.

(c) If the out-of-balance mass is estimated to be 1.5 % of the total mass, estimate the value of e.

5.[6] m, k, c F\_0 가 가 (steady-state response)

$$X = \frac{F_0}{k} \frac{1}{\sqrt{(1-r^2)^2 + (2\zeta r)^2}}$$

$\omega_n$  ,  $\zeta$  X F\_0/k

(a)  $\zeta$ 가 0, 0.5, 0.7  $\bar{X}$  r

(b) (a) , m, c, k

(c) (transient response) 가

1.[2 ] 가  
가

2.[6 ] O , X ( )

(a) 440 Hz 가  
(La) 0.2  
(beat)  
(La) 0.2  
(La)

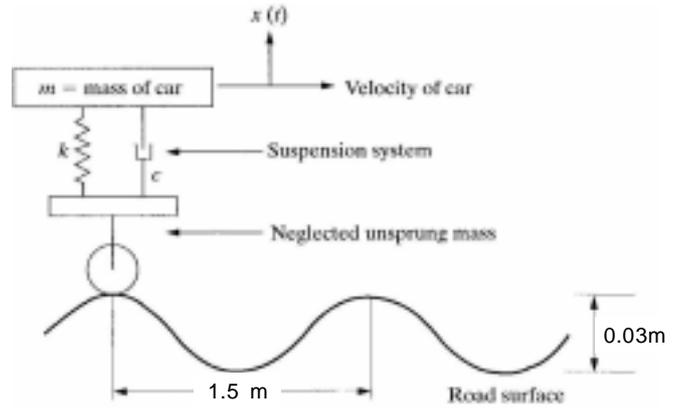
0.1 ( )  
:  
(b)

(c) 가  
( )

3.[3 ]  $k = 15000 \text{ N/m}$ ,  $m = 5 \text{ kg}$ ,  $F(t) = 800 \sin(81.6t) \text{ N}$  가  
0.01 m,  
20 m/s

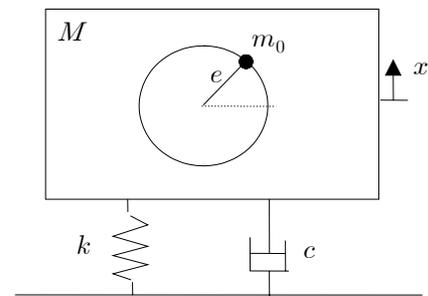
4.[3 ] 1 가  
$$\frac{X_b}{Y} = \frac{X_b/Y}{\sqrt{(1-r^2)^2 + (2\zeta r)^2}}$$
  
 $\zeta$ 가 0, 0.5, 0.7  
 $X_b/Y$  r

5.[6 ] Consider a single-degree-of freedom model of an automobile driving over a road as shown below. The road surface is approximated as sinusoidal in cross-section providing a base motion displacement of  $y(t) = Y \sin \omega_b t$ . The mass of the automobile is 750 kg, and its speed is 20 km/h.



- (a) Determine the values of  $Y$  and  $\omega_b$ .
- (b) Determine the amplitude  $X_b$  of the automobile vibration when the undamped natural frequency  $\omega_n$  is 13.5 rad/s and damping ratio  $\zeta$  is 0.180.
- (c) Determine the magnitude of the force transmitted to the car through the suspension system, if the amplitude of the automobile vibration is 0.0319 m.

6.[6 ] M  
 $m_0$  가 e  
1 N



- (a) M (free-body diagram)
- (b) N 1200 rpm, M 10 kg,  $m_0$ 가 0.2 kg, e 0.05 m, k가 15,000 N/m, c가 300 kg/s mm 가?
- (c) (b)

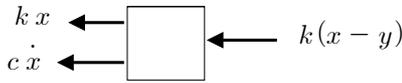
1. ([ ] : , , . : , , .)

2. (a)  $X \quad T_b = \frac{2\pi}{|\omega_n - \omega|}$  ,  $|\omega_n - \omega|$  가  $T_b$  가

(b)  $X \quad \frac{X_b}{Y}$  ,  $m \approx 0$   $\omega_n$   $r \approx 0$   $\frac{X_b}{Y} \approx 1$

(c)  $O$  (  $X_b/Y$  ) ,  $k$   $\omega_n$   $r$   $\zeta$  가 .

3. (a)



$$-c \dot{x}(t) - kx(t) - k[x(t) - y(t)] = m \ddot{x}(t)$$

$$m \ddot{x}(t) + c \dot{x}(t) + 2kx(t) = ky(t)$$

(b)  $y(t) = Y \sin(20\pi t)$  mm  $x(t) = X \sin(20\pi t - \theta)$

$$F_{tr}(t) = kx(t) + c \dot{x}(t) = kX \sin(20\pi t - \theta) + c(20\pi)X \cos(20\pi t - \theta)$$

$$F_T = \sqrt{(kX)^2 + [c(20\pi)X]^2} = \sqrt{(k)^2 + [c(20\pi)]^2} X$$

$$= \sqrt{(12000 \text{ N/m})^2 + [(3400 \text{ Ns/m})(20\pi \text{ rad/s})]^2} (6.0 \text{ mm}) = (214000 \text{ N/m})(6.0 \times 10^{-3} \text{ m}) = 1284 \text{ N}$$

4.  $m = 120 \text{ kg}$ ,  $k = 500 \text{ kN/m}$ ,  $c = 4000 \text{ kg/s}$ ,  $N = 3600 \text{ rpm}$ ,  $F_0 = 350 \text{ N}$

(a)  $\omega_n = \sqrt{\frac{500 \times 10^3 \text{ kg/s}^2}{120 \text{ kg}}} = 64.5 \text{ rad/s}$ ,  $\omega_r = \frac{(2\pi \text{ rad})(3,600 / \text{min})}{60 \text{ s/min}} = 377 \text{ rad/s}$

$$r = \frac{377}{64.5} = 5.84, \quad \zeta = \frac{4000 \text{ kg/s}}{2\sqrt{(120 \text{ kg})(500 \times 10^3 \text{ N/m})}} = 0.258$$

(b)  $X = \frac{m_0 e}{m} \frac{\omega_r^2}{\sqrt{(\omega_n^2 - \omega_r^2)^2 + (2\zeta\omega_n\omega_r)^2}} \quad (m_0 e \omega_r^2 = F_0)$

$$= \frac{F_0}{m \omega_r^2} \frac{r^2}{\sqrt{(1-r^2)^2 + (2\zeta r)^2}}$$

$$= \frac{(350 \text{ N})}{(120 \text{ kg})(377 \text{ rad/s})^2} \frac{5.84^2}{\sqrt{(1-5.84^2)^2 + [2(0.258)(5.84)]^2}} = (2.052 \times 10^{-5} \text{ m})(1.026)$$

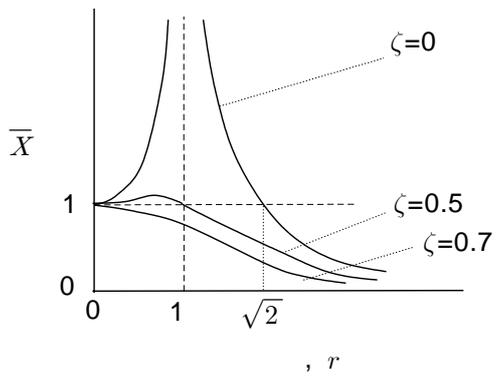
$$= 2.105 \times 10^{-5} \text{ m} = 0.0211 \text{ mm}$$

(c)  $\frac{m_0}{m} = 0.015$

$$m_0 = 0.015 m = 0.015 (120 \text{ kg}) = 1.80 \text{ kg}$$

$$e = \frac{F_0}{m_0 \omega_r^2} = \frac{350 \text{ N}}{(1.80 \text{ kg})(377 \text{ rad/s})^2} = 1.368 \times 10^{-3} \text{ m} = 1.368 \text{ mm}$$

5. (a)



(b) 1.

2. 가

가

,  $c$  ,  $m$  ,  $k$  .

(c) (transient response)

(steady-state response)

, .

1. ( : (vortex), 가 ),

: , , ,

2. (a) O  $\frac{2\pi}{|\omega_1 - \omega_0|} = 0.2 \text{ s}$   $\omega_1 - \omega_0 = \pm \frac{2\pi \text{ rad}}{0.2 \text{ s}}$

가  $\omega_2 - \omega_0 = \pm \frac{2\pi \text{ rad}}{0.2 \text{ s}}$

$(\omega_1 - \omega_0) - (\omega_2 - \omega_0) = \frac{2\pi \text{ rad}}{0.2 \text{ s}} - \left( \pm \frac{2\pi \text{ rad}}{0.2 \text{ s}} \right) = 0$   $\pm \frac{4\pi \text{ rad}}{0.2 \text{ s}}$

,  $\omega_1 - \omega_2 = \pm \frac{4\pi \text{ rad}}{0.2 \text{ s}}$ ,  $\frac{2\pi}{|\omega_1 - \omega_2|} = 0.1 \text{ s}$

(b) O  $\frac{X_b}{Y}$ ,  $r \sqrt{2}$ ,  $k$   $r$

,  $\zeta$ .

(c) X  $m X_r / m_0 e$ ,  $\omega_r$   $\omega_n$   
 $\omega_r$ .

3.  $k = 15000 \text{ N/m}$ ,  $m = 5 \text{ kg}$ ,  $F(t) = 800\sin(81.6t) \text{ N}$ ,  $x_0 = 0.01 \text{ m}$ ,  $v_0 = 20 \text{ m/s}$

$\omega_n = \sqrt{\frac{15000 \text{ N/m}}{5 \text{ kg}}} = 54.8 \text{ rad/s}$ ,  $f_0 = \frac{800 \text{ N}}{5 \text{ kg}} = 160 \text{ m/s}^2$

$\frac{f_0}{\omega_n^2 - \omega^2} = \frac{(160 \text{ m/s}^2)}{(54.8 \text{ rad/s})^2 - (81.6 \text{ rad/s})^2} = -0.0437 \text{ m}$ ,  $X = 0.0437 \text{ m}$

[ 1 ]

$x(t) = A_1 \sin\omega_n t + A_2 \cos\omega_n t - X \sin 81.6t$

$x(0) = A_2 = x_0$   $A_2 = 0.01 \text{ m}$

$\dot{x}(0) = \omega_n A_1 - (81.6)X = v_0$   $A_1 = \frac{v_0 + 81.6 X}{\omega_n} = \frac{(20 \text{ m/s}) + (81.6 \text{ rad/s})(0.0437 \text{ m})}{54.8 \text{ rad/s}}$   
 $= 0.430 \text{ m}$

$x(t) = 0.430 \sin 54.8t + 0.01 \cos 54.8t - 0.0437 \sin 81.6t \text{ m}$

[ 2 ]

$x(t) = A \sin(54.8t + \phi) - (0.0437 \text{ m}) \sin 81.6t$

$x(0) = A \sin\phi = (0.01 \text{ m}) > 0 \dots$

$\dot{x}(0) = (54.8 \text{ rad/s}) A \cos\phi - (0.0437 \text{ m})(81.6 \text{ rad/s}) = 20 \text{ m/s}$

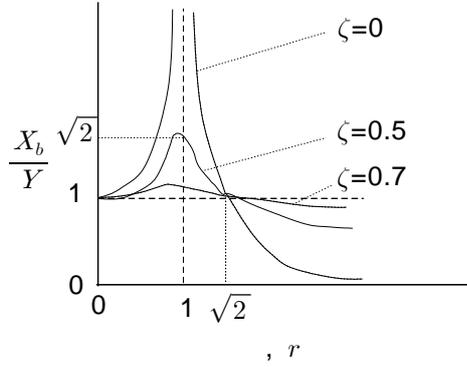
$A \cos\phi = 0.430 \text{ m} > 0 \dots$

$A = \sqrt{(0.01 \text{ m})^2 + (0.430 \text{ m})^2} = 0.430 \text{ m}$

$\phi = \tan^{-1} \frac{0.01}{0.430} = 0.023 \text{ rad}$

$x(t) = 0.430 \sin(54.8t + 0.023) - 0.0437 \sin 81.6t \text{ m}$

4.



5. (a)  $Y = \frac{1}{2}(0.03 \text{ m}) = 0.015 \text{ m}$

$$f = (20 \text{ km/h}) \left( \frac{1 \text{ cycle}}{0.0015 \text{ km}} \right) \left( \frac{1 \text{ hour}}{3600 \text{ s}} \right) = 3.70 \text{ Hz}, \quad \omega_b = (3.70 \text{ Hz}) \left( \frac{2\pi \text{ rad}}{1 \text{ cycle}} \right) = 23.3 \text{ rad/s}$$

(b)  $r = \frac{\omega_b}{\omega_n} = \frac{23.3 \text{ rad/s}}{13.5 \text{ rad/s}} = 1.726, \quad \zeta = 0.180$

$$X_b = Y \frac{\sqrt{1 + (2\zeta r)^2}}{\sqrt{(1 - r^2)^2 + (2\zeta r)^2}} = (0.015 \text{ m}) \frac{\sqrt{1 + [2(0.18)(1.726)]^2}}{\sqrt{[1 - (1.726)^2]^2 + [2(0.18)(1.726)]^2}}$$

$$= (0.015 \text{ m})(0.568) = 0.00851 \text{ m} = 8.51 \text{ mm}$$

(c)  $F_{tr}(t) = k(x - y) + c(\dot{x} - \dot{y})$

$$= -m\ddot{x} = m\omega_b^2 X_b \cos(\omega_b t - \theta_b - \phi) = F_T \cos(\omega_b t - \theta_b - \phi)$$

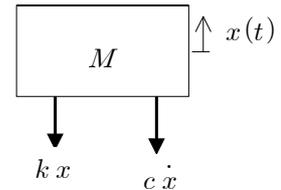
$$F_T = m\omega_b^2 X_b = (750 \text{ kg})(23.3 \text{ rad/s})^2 (0.0319 \text{ m}) = 13,000 \text{ N} = 13.0 \text{ kN}$$

6. (a)  $-kx - c\dot{x} = (M - m_0)\ddot{x} + m_0 \frac{d^2}{dt^2}(x + e \sin \omega_r t)$

$$= M\ddot{x} - m_0\ddot{x} + m_0\ddot{x} - m_0 e \omega_r^2 \sin \omega_r t$$

$$M\ddot{x} + c\dot{x} + kx = m_0 e \omega_r^2 \sin \omega_r t$$

$$M\ddot{x} + c\dot{x} + kx = m_0 e \left( \frac{2\pi N}{60} \right)^2 \sin \frac{2\pi N}{60} t$$



(b)  $\omega_r = (2\pi \text{ rad})(1,200 \text{ 1/min}) \left( \frac{1 \text{ min}}{60 \text{ s}} \right) = 127.5 \text{ rad/s}$

$$\omega_n = \sqrt{\frac{k}{M}} = \sqrt{\frac{15,000 \text{ N/m}}{10 \text{ kg}}} = 38.7 \text{ rad/s}, \quad r = \frac{\omega_r}{\omega_n} = \frac{127.5}{38.7} = 3.25$$

$$\zeta = \frac{c}{2\sqrt{Mk}} = \frac{300 \text{ kg/s}}{2\sqrt{(10 \text{ kg})(15,000 \text{ N/m})}} = 0.387$$

$$X_r = \frac{m_0 e}{M} \frac{r^2}{\sqrt{(1 - r^2)^2 + (2\zeta r)^2}} = \frac{(0.2 \text{ kg})(0.05 \text{ m})}{10 \text{ kg}} \frac{(3.25)^2}{\sqrt{(1 - 3.25^2)^2 + [2(0.387)(3.25)]^2}}$$

$$= (0.001 \text{ m})(1.068) = 0.001068 \text{ m} = 1.068 \text{ mm}$$

(c)  $F_{tr}(t) = kx + c\dot{x} = kX_r \sin(\omega_r t - \theta_r) + c\omega_r X_r \cos(\omega_r t - \theta_r) = F_T \sin(\omega_r t - \theta_r + \phi)$

$$F_T = \sqrt{(kX_r)^2 + (c\omega_r X_r)^2} = \sqrt{k^2 + (c\omega_r)^2} X_r$$

$$= \sqrt{(15,000 \text{ N/m})^2 + [(300 \text{ kg/s})(125.7 \text{ rad/s})]^2} (0.001068 \text{ m}) = 43.3 \text{ N}$$